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ملخص :

مختبر مقاومة المواد

Strength Lab

للطالب : أنس البرغوث

اللجنة الأكاديمية لقسم الهندسة الصناعية

2023



LAB strength ((Mid))

Anas Barghouth

- Tensile test
- Compression test
- Torsion test
- stability of column test
- Deflection of Beam test
- Impact test
- strain gauge test

① Tensile Test

المحول: يقيس القوة
Alea: تقبل
عينة

⇒ UTM

"universal Testing Machine"

two part → loading
measuring

نظري

ductile (steel)

brittle (brass, concrete)

شكل الكسر

cup and cone
45°

شكل الكسر

plat
90°

السبب

shear stress

ult

المتانة

المتانة

fra

السبب

Normal stress

ult = fra = yield

E

E

low

high

Carbonic steel
deformation
"strain"

كبير

أقل

العمل

6. E

بشبكة

الرسم

طريقة العمل

نفا → (6. E)

لا → (F, δ)

(F/A₀, δ/L₀)

proportional limit → نهاية الخط المستقيم

yeild stress point

offset

avg

0.001

up+down

2

ultimate → أعلى نقطة

Fracture → نهاية الرسم

نظام مينة



القوانين

Hooke's Law

$$\sigma = E \epsilon$$



$$E = \text{slope} = \frac{\sigma}{\epsilon}$$

F , work done

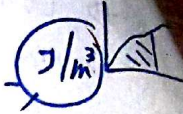
P , work done

work done \rightarrow يقضي الطاقة
energy dissipated

modulus of elasticity

modulus of Resilience

$$U_R = \frac{1}{2} \sigma_y \epsilon_y$$



modulus of Toughness

$$U_T = \frac{2}{3} \sigma_{max} \epsilon_{max}$$



$$\% \text{ elongation} = \left| \frac{l_f - l_o}{l_o} \right| \times 100\% = \epsilon_{max} \times 100\%$$

$$\text{Reduction of Area} = \left| \frac{A_o - A_f}{A_o} \right| \times 100\%$$

to find A_f :-

$$A_f = \frac{A_o}{1 + \epsilon_{max}}$$

Area at fracture

$A_f l_f = A_o l_o$ (plastic region)

bulk modulus of elasticity

$$k = \frac{E}{3(1-2\nu)}$$

shear

$$G = \frac{E}{2(1+\nu)}$$

$$\nu = 0.33$$

$\epsilon_{lateral}$
 ϵ_{axial}

حالة حرة

True actual

$$\epsilon_T = \ln(1 + \epsilon)$$

$$\sigma_T = \sigma(1 + \epsilon)$$

\rightarrow

% elongation \rightarrow

ductile
> 5%

brittle
< 5%

\sim شدة

uniaxial Load

$$\nu = \frac{-\epsilon_{lateral}}{\epsilon_{axial}} = \frac{-\text{reduction of area}}{\text{elongation}} = \frac{\left| \frac{d_f - d_o}{d_o} \right|}{\left| \frac{l_f - l_o}{l_o} \right|}$$

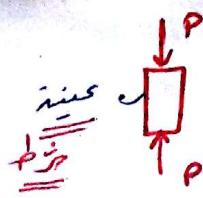
reduction of area

$$\left| \frac{d_f - d_o}{d_o} \right|$$

elongation

② Compression test

Buckling تجنب $D > L$



- الطول، يقل
- Area: تزيد

⇒ UTM

Q₁: brittle يتأثر بـ Comp. أكثر من ten. : ليس عملنا التجريبية

Q₂: شو الفرق بين Comp. ten. :
- yield - نفسو
- ult. - يختلف

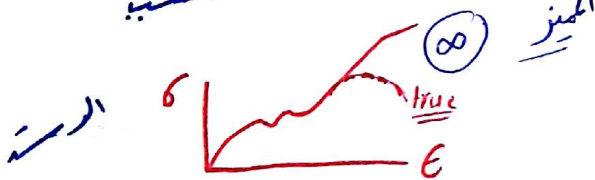
النظري

ductile

شكل الكسر

no fracture

السبب

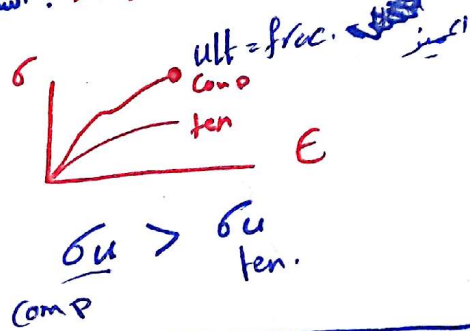


brittle

شكل الكسر



السبب: Normal stress



Note :-

①

lubricate (oil)

② To less the friction (so we have uniform inc ↑ in Area)

③ To avoid Barrel shape

③

concrete (comp)
steel (Ten.)

③ ضايق
brittle - comp.
ductile - Ten.

④

الفرغ

كل عينه فيها شوائب
Ten - بأثر
Comp - ما بأثر

load يتحمل أكبر

⑤

sample المستخدمة

$D > L$



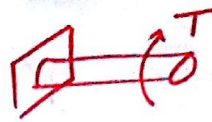
- 1/4 eeee 1

To avoid Buckling

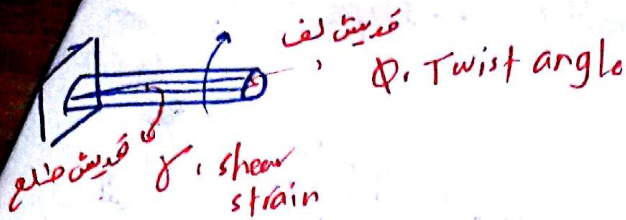
الصل

tensile test - نفس

③ Torsion test



undercut
every cross-section → remain plane and rotates as a rigid disc



$$\gamma = \frac{\Phi r}{L}$$

دائراً
تدور

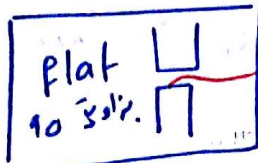
$$\Phi = \frac{TL}{JG}$$

$$\gamma = \frac{Tr}{JG}$$

نظري

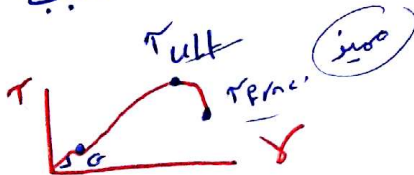
ductile

شكل الكسر



ناعم

السبب: shear stress



brittle

a) pure

(الطباشير)

شكل الكسر



قاسم

السبب Normal stress

b) not pure — ductile
(brass) ult = prac = yield

اعين



انها

$$\tau, \gamma$$

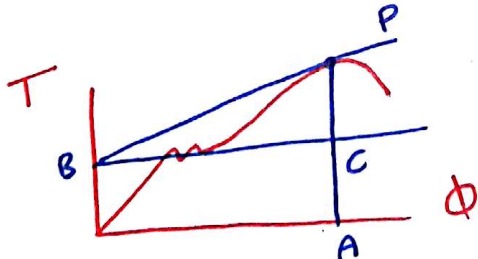
بشيت

الحل

الرسم

غير ذلك

⇒



shear strain ⇒ $\gamma = \frac{\Phi r}{L}$

shear stress

الاجل

elastic

plastic

باخذ محاس ونزل المحور

$$\tau = \frac{TC}{J} = \frac{\pi}{32} (d^4 - d^4)$$

المحور

المحور

المحور

$$\frac{dT}{d\Phi} = \frac{PC}{\Phi}$$

$$\tau = \frac{PC + 3PA}{2\pi r^3}$$

المحور

$$\tau = \frac{1}{2\pi r^3} \left[\Phi \frac{dT}{d\Phi} + 3T \right]$$

المحور

المقاومة

Hooke's Law $\tau = G \gamma$ — rad

→ modulus of rigidity $G = \text{slope} = \frac{\tau}{\gamma}$

→ modulus of resilience $U_r = \frac{1}{2} \tau_y \gamma_y$

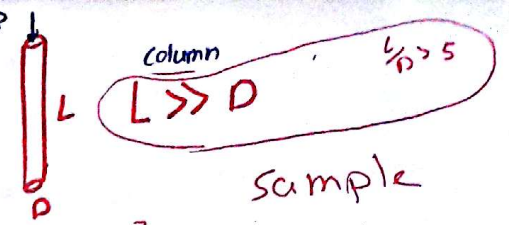


→ modulus of rupture (Toughness) $U_T = \frac{2}{3} \tau_{\max} \gamma_{\max}$



④ stability of column

1- P_{cr} : The load at which the Buckling will occur



2- y : Max deflection

lateral deflection

[بسبب فشل قبل الوصول Normal strength]
دلالة

نظري

stable

$$P < P_{cr}$$

التواء طفيف وجود
فشل

unstable

$$P \geq P_{cr}$$

Buckling failure

زاوية انحناء slope

$$P = m \left(\frac{P}{y} \right) + P_{cr}$$

الحل

البيانات

P (N)	y (mm)
...	...

الرجوع

بوصلة ①

بيانات ②

P

P_{cr}

P/y

$$P/y = 0 \Rightarrow y = \infty$$

$\Rightarrow (P_{cr})$ (y-intercept)

$$\frac{\pi^2 E I}{32 D^4}$$

ن: عدد الانحناءات
وغالباً $l =$

المواد المركبة
Buckling
دائري بغير قوسين

Euler formula

k

l_e = length of half sine wave

$$P_{cr} = \frac{n^2 \pi^2 E I}{(kL)^2}$$

effective length

fixed-fixed

$$k = 0.5$$



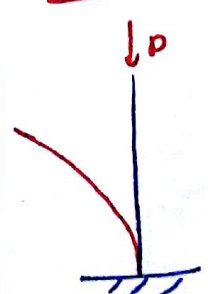
Fixed-pin

$$k = 0.7$$



fixed-free

$$k = 2$$



Note

(slenderness ratio (L/k)) radius of generation \rightarrow related to the value of critical force

5) Deflection of Beam (at elastic zone)

نظري

a - الأجهزة → Dynamometer ⇒ Reaction (N)
 → Dial gauge ⇒ deflection (mm)

الموقع: نفس مكان Force
 القراءة: كل لفنة كابل 1mm وكدفنة 100 P
 Ex: 3 لفات = 3mm = 3.07

b - Notes

1) لا ننسى وزن Beam ⇒ مصفر عند التثبيت (Force عند free end)
 2) Max deflection

المحل - حساب deflection

Reaction

$$R_B = \frac{Fx}{L}$$

$$R_A = F(1 - \frac{x}{L})$$

deflection

$$0 < x < \frac{L}{2} \quad w(x) = \frac{FL^3}{48EI} \left[\frac{3x}{L} - \frac{4x^3}{L^3} \right]$$

$$x = \frac{L}{2} \quad w(x) = \frac{FL^3}{48EI}$$

$\sum M_A = 0$
 $\sum F_y = 0$

2) cantilever

deflection كبير مما قد يؤدي إلى تلف الجهاز وتلف Dial gauge

$w(x) = \frac{FL^3}{3EI}$ (where load is exist)

المحل - موقع Force

$w(x) \rightarrow w'(x) = 0$
 $x =$

② Impact test

وحدود

Impact load
(shock ")

زمن التأثير
قليل جداً

⇒

$\Delta t < \frac{1}{3}$ natural freq. of the Material

الهدف حساب energy absorbed ⇒ (Toughness)

energy لا يتجزئها
العينة لقد اكسر

ductility + Geometry ← تعتمد على

Note → لا تستخدم الفيت Design باد notch

نظري

① Notch — $\begin{matrix} \rightarrow V \\ \rightarrow U \end{matrix}$ $\begin{matrix} \text{كبير} \\ \text{قليل} \end{matrix}$ $\begin{matrix} \text{عالية جداً} \\ \text{قليلة جداً} \end{matrix}$

الهدف → تسهيل عملية تكسر \rightarrow عن طريق \rightarrow inc ↑ stress concentration

نقطة Notch ركوة على \rightarrow tension

	Izod	Charpy (أنواع الشارة)
support	cantilever	simply support Beam
اتجاه أو Load بالمنطقة Notch	same side	opposit side
Toughness	أقل ~ أكثر شكس شرك	أكبر ~ أقل شكس شرك
	[one shearing Area]	[two shearing Area]



$$E_{Izod} = \frac{1}{2} E_{Charpy}$$

③ تأثير $temp$ [Temp & toughness]

السبب $\Rightarrow \uparrow temp \rightarrow \uparrow ductility \rightarrow \uparrow toughness$

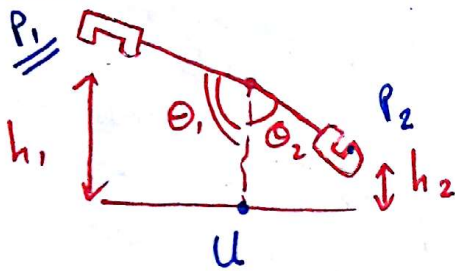
④ التكملة
مفصلتين

① بدون عينة \rightarrow friction

② عينة \rightarrow قراءه

$\Rightarrow toughness = القراءه - friction$

الحل



$$P_1 = U + P_2 + \text{friction}$$

$$\therefore U = \frac{P_1}{mgh_1} - \frac{P_2}{mgh_2} - \text{friction}$$

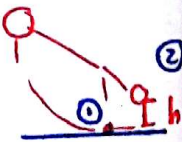
حالتين
 h_1, h_2
 l, θ

$$U = mg(h_1 - h_2) - \text{friction}$$

$$U = mgl(\cos\theta_1 - \cos\theta_2) - \text{friction}$$

حالة خاصة $\rightarrow h_2$, Max Velocity

at $h=0$



energy cons.

$$E_1 = E_2$$

$$\frac{1}{2}mv^2 = mgh_2$$

$$v_2 = \frac{v^2}{2g}$$

Ex1: which higher toughness ① al m $\uparrow ductility$
المنطقه ② Geomtry ③ steel

Ex

Ex2:- rank toughness \downarrow

steel u, steel v
brass v, brass u

① ductility ② Geomtry

steel \rightarrow u ①
v ②
brass \rightarrow u ③
v ④

⑥ strain gauge [strain بقيت]

PeZbo electric materials

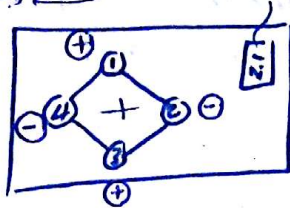
مواد حامية
جدا للغير
المقاومة الكهربائية

- 4- resistores
- 2- channel

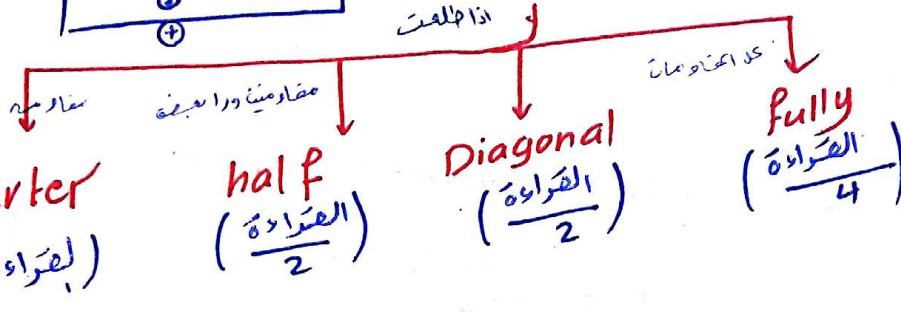
نظري

- wheatstone Bridge

الفراة (Voltage) يستعمل

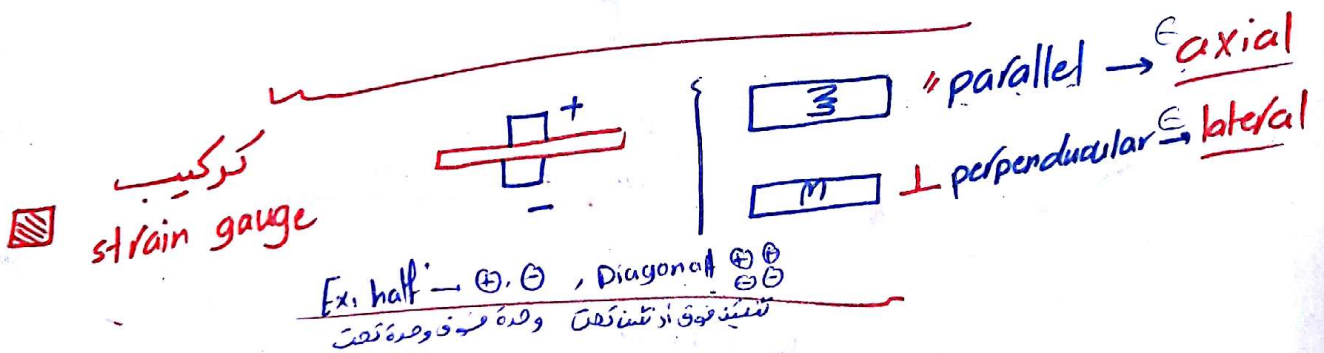


strain gauge
مقاومة بطولها من

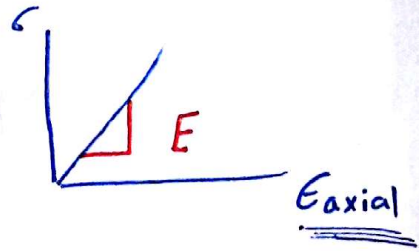
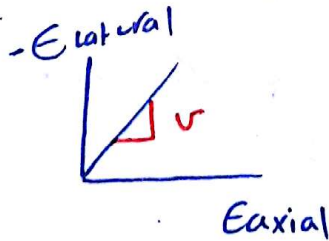
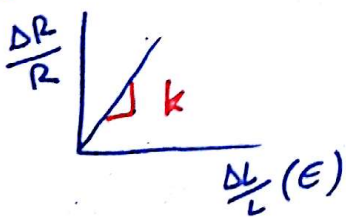


strain
الفعلي

=



الاسمات



k: sensitivity Factor

معلومات
معمارية

$$\frac{\Delta R}{R} = k \left(\frac{\Delta L}{L} \right) \epsilon, \quad \frac{V_{out}}{V_{in}} = \frac{1}{4} \left[\frac{\Delta R_1}{R_1} - \frac{\Delta R_2}{R_2} + \frac{\Delta R_3}{R_3} - \frac{\Delta R_4}{R_4} \right], \quad R_s = \frac{PL}{A}$$

العمل

$$\textcircled{1} \quad \nu = - \frac{\epsilon_{\text{lateral}}}{\epsilon_{\text{axial}}}$$

E_{x1} parallel = 0.2
perpendicular = -0.1

$$\epsilon_{\text{axial}} = 0.2$$

$$\epsilon_{\text{lateral}} = -0.1$$

$$\nu = \dots$$

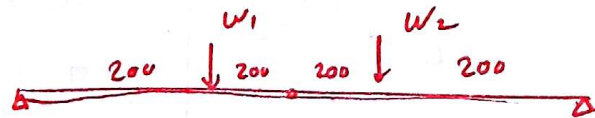
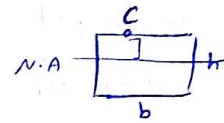
E_{x2} perpendicular (full) = 0.12
parallel (quarter) = 0.7

$$\epsilon_{\text{lateral}} = \frac{0.12}{4} = 0.3$$

$$\epsilon_{\text{axial}} = 0.7$$

② Max bending stress

$$\sigma = \frac{MC}{I}$$



① Reaction

② \rightarrow Max. Moment
shear-Moment

طبق
القانون

نطلب عند نقطة معينة \rightarrow Reaction
2) section — moment
 \Rightarrow طبق القانون

Tensile

Proportional Limit :- Is the point on the curve bet. σ & ϵ where it deviates first time from a straight line

2] Elastic Limit :- The max load that can be applied to the specimen without Permanently deformation.

3] Yield stress :- The elongation occurs for the first time without incⁿ in load.

4] Ultimate stress :- σ_u the max stress that can be applied before fracture. ^{value that represent}

5] Modulus of resilience :- (U_R) is the amount of energy stored in stressing the material to the Elastic Curve.

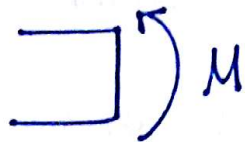
6] Modulus of Toughness :- (U_T) The total energy absorption capabilities of the material to failure.

7] The ductility of Material :- It is the ^{load} ability of material to deform under \Rightarrow indicated from percentage in :-
1) elongation of length
2) reduction of Area

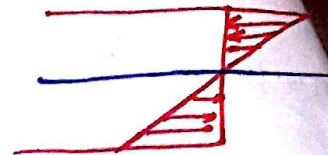
$$\% \text{ Error} = \left| \frac{\text{Th} - \text{exp}}{\text{Th}} \right| \times 100 \%$$

Bending stress

[1]



$$\sigma = \frac{MC}{I}$$

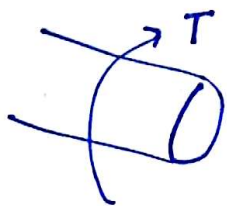


Max (outer surface)

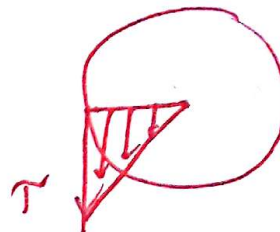
Min = Zero (N.A)

Torsion

[2]



$$\tau = \frac{Tr}{J}$$

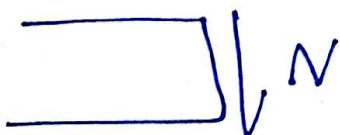


Max (outer surface)

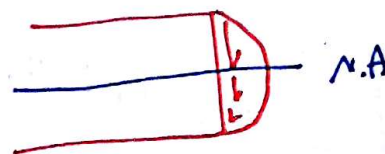
Min = Zero (center)

shear stress

[3]



$$\tau = \frac{VQ}{Ib}$$



Max (N.A)

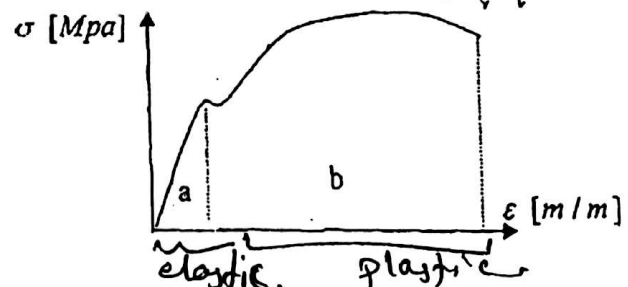
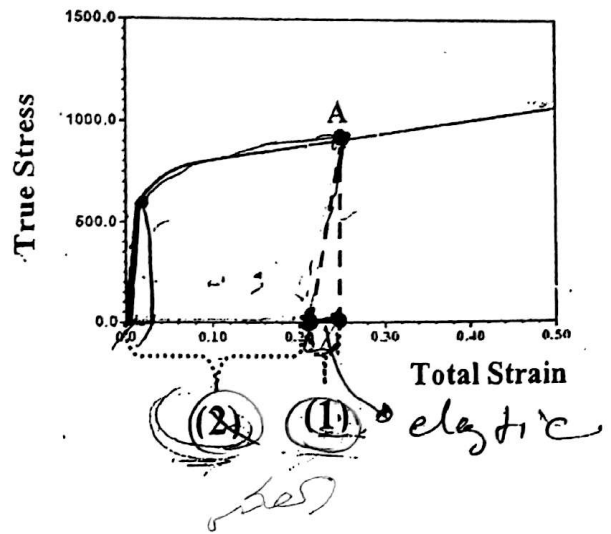
Min = Zero (outer surface)

STUDENT NAME: محمد أحمد العناني STUDENT No.: 631372 Inst. Name: د. هادي تليان

Consider the following tension stress - strain diagram for ductile material.

(Answer items 1 & 2)

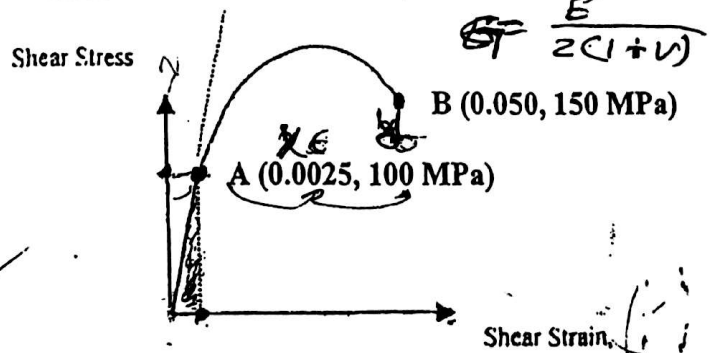
- The elastic and plastic strain of point A are
 - 1 & 2, respectively
 - 2.
 - 2 & 1, respectively
 - The whole strain axis.
- If the fracture occurred at $\epsilon = 0.5$, The ductility of the material is
 - 10.0 μm
 - 50.0 %
 - 5.0 μm
 - 0.5 %
- In the strain gauge experiment both axially and laterally oriented strain gauges were attached to simply supported beam to be able to estimate experimentally.
 - The modulus of elasticity E .
 - The shear modulus G .
 - The Poisson's ratio ν .
 - Longitudinal and radial strains.
- The strain gauge technique is used to measure
 - Stress.
 - Deflection.
 - Resistance.
 - Strain.
- In the strain gauge, small change in dimensions are translated into equivalent change in:
 - Resistance.
 - Conductivity
 - Sensitivity
 - Heat
- In the typical tensile test curve shown aside, the area under the curve is the energy absorbed by the test specimen per unit volume, the modulus of toughness is
 - Area a
 - Area a + Area b
 - Area b.
 - Area b - Area a
- Poisson's ratio is defined as
 - The ratio of lateral stain to axial strain
 - The ratio of change of the axial length to the change of the transverse length when the element is
 - The ratio of lateral stain to axial strain when the member is loaded just axially
 - b and c above are correct.



Consider the following torsion shear stress - shear strain diagram for ductile material.

(Answer 8 & 9)

- The modulus of rigidity G is
 - 40.0 MPa.
 - 50.0 MPa
 - 100 GPa
 - 40.0 GPa.



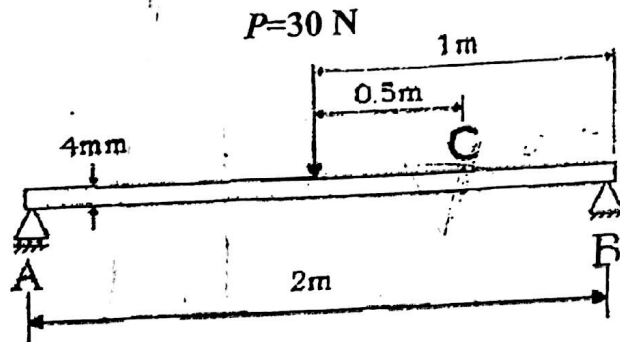
- The modulus of Resilience is
 - 0.125 MPa.
 - 0.125 GPa.

- 0.250 GPa
- 0.250 MPa.

$$\frac{1}{2} \times 100 \times 0.0025 = 0.125$$

$$E = \frac{6.0025 - 0}{100 - 0}$$

A rectangular cross sectional area (20mm x 4 mm) is simply supported and loaded as shown.
(Answer items 10-14)



10. The reaction at support A and B is:
 a) 30 and 30 N
 b) 15 and 15 N
 c) 15 and 30 N
 d) 30 and 15 N
11. The maximum deflection occurs at point:
 a) B
 b) A
 c) Center of the beam
 d) C
12. The moment at point C is equal to:
 a) 15.0 N.m
 b) 10.0 N.m
 c) 30.0 N.m
 d) 7.5 N.m
13. If the beam is replaced by another one of same material, same length and same loading but with circular cross sectional area of diameter 10 mm, then:
 a) Beam of rectangular cross section will deflect more than circular one.
 b) Beam of circular cross section will deflect more than rectangular one.
 c) Both beams have same deflection.
 d) Cannot be determined
14. The stress at point C is equal to:
 a) 14.063 MPa
 b) 140.63 MPa
 c) 9.75 Mpa
 d) 18.75 MPa
15. For a column subjected to a compressive load. If the length of the column is reduced by one half, the critical buckling load will be
 a) Reduced by one half
 b) Doubled
 c) Increase 4-times
 d) The same
16. One off the following is an example for buckling of column:
 a) Walking stick
 b) Column in a building
 c) A beam loaded by an axial compressive force only
 d) All of above
17. For the same specimen size, compression test usually requires _____ energy per unit volume than that tensile test
 a) More
 b) Less
 c) Equal
 d) Don't know
18. To reduce or eliminate barreling in compression test, we must
 a) Increase the load
 b) Decrease the friction
 c) Decrease the load
 d) Increase the friction
19. For the specimen shown the mode under which failure took place for ductile material is
 a) Tensile load
 b) Compressive load
 c) Torsional Load
 d) Buckling
20. In the tension test, the property which is an indication of the stiffness of a material is:
 a) Ultimate strength
 b) Elastic limit
 c) Proportional limit
 d) Modulus of elasticity

Good Luck

$$A + L_A = A_0 L_0$$

$$A_f = \frac{A_0}{1 + \epsilon_{max}}$$

Usually find
but can give it in mid

Find Q_R

Elastic

$$N = -\frac{\epsilon_{extension}}{\epsilon_{contraction}}$$

$$\frac{Q_0 - Q_0}{Q_0}$$

$$\frac{L_0 - L_0}{L_0}$$

Plastic

$$A_f = \frac{A_0}{1 + \epsilon_{max}}$$

~~elastic~~

① higher E

② most ductile

[3.5 points] 1. The engineering stress-strain curves of three materials A, B, and C obtained from a tensile test are shown in Fig. 1. Answer Qs. (1-1 ~ 1-10):

1-1. The material... A ... has the highest elastic modulus.

1-2. The material... C ... is the most ductile.

1-3. The material... A ... is most brittle.

1-4. The material... B ... has the highest tensile strength.

1-5. The material... C ... has highest toughness.

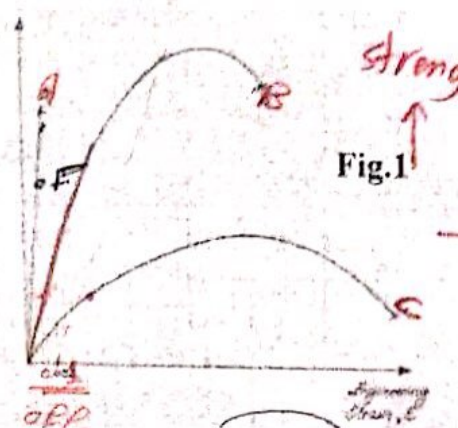
1-6. The material... A ... has highest resilience.

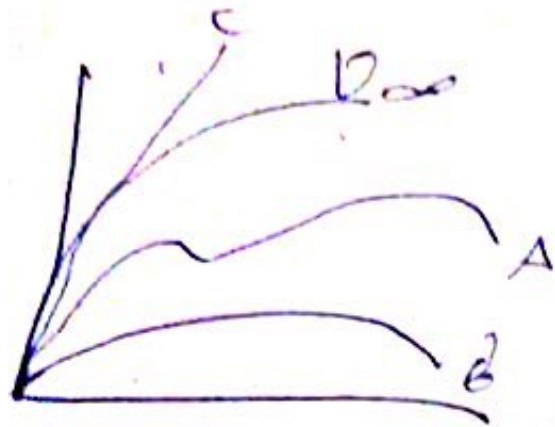
1-7. The material... A ... can carry the highest load without permanent deformation.

[10 points] 2. For the tensional bar given in Fig. 2 answer Qs.

(2-1 ~ 2-10)

2-1. The reaction at the support is..... 800 N



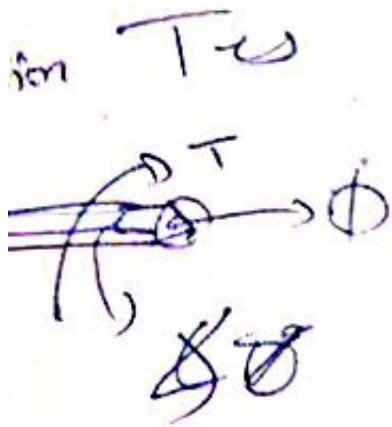


If that Tensile test
What is ductile

ductile A, B, D

Brittle C

true D



If that can't

ductile A, B, D

Brittle C

true A, B

* Back to Page of
[Capt test] and
distinguish between
True & Engineering
graphs

$$\frac{TL}{JG}$$

(- d_i)
and inner

$$= \frac{\Phi r}{L}$$

$$S = \frac{Tr}{JG}$$

Remember it

Two steel shaft $G = 80 \text{ GPa}$ $D_1 = 20 \text{ mm}$ $D_2 = 10 \text{ mm}$
 under the same applied Torque find $\frac{\gamma_1}{\gamma_2} ??$

$$\gamma = \frac{\phi r}{L}$$

$$= \frac{Tr}{JG}$$

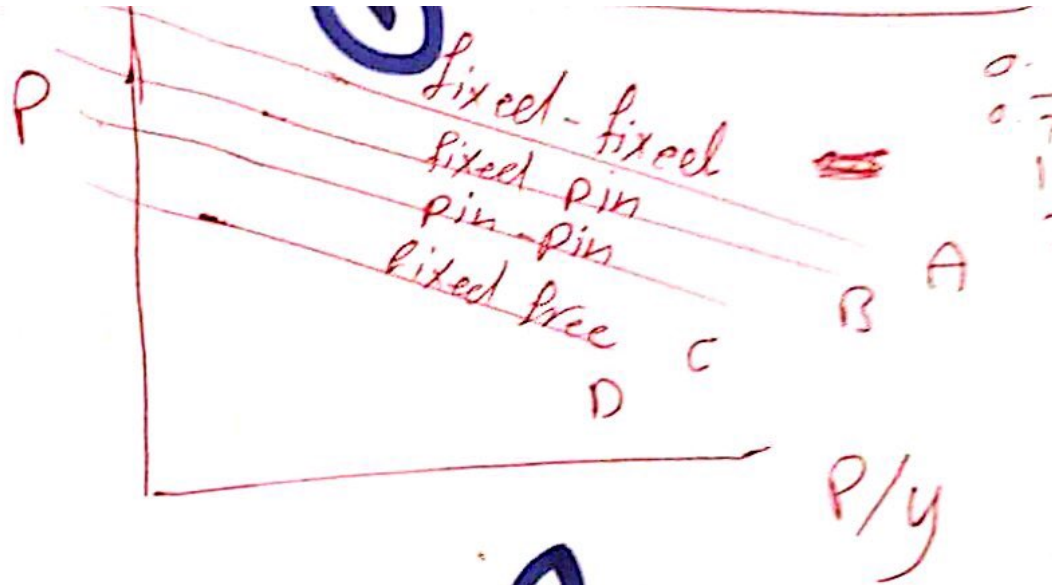
$$\frac{\gamma_1}{\gamma_2} = \frac{\frac{Tr}{JG}}{\frac{Tr}{JG}} = \frac{\frac{r_1}{J_1}}{\frac{r_2}{J_2}} = \frac{\frac{P_1}{J_1}}{\frac{P_2}{J_2}}$$

$$= \frac{\frac{P_1}{\cancel{\frac{\pi}{32}} D_1^4}}{\frac{P_2}{\cancel{\frac{\pi}{32}} D_2^4}} = \frac{\left(\frac{1}{D_1^3} \right)}{\left(\frac{1}{D_2^3} \right)} = \frac{D_2^3}{D_1^3}$$

$$= \underline{\underline{0.125}}$$

5

$$\uparrow P_{cr} = \frac{k}{\downarrow}$$



fixed-fixed
fixed-free
pin-pin
pin-fixed

A
D
C
B

①

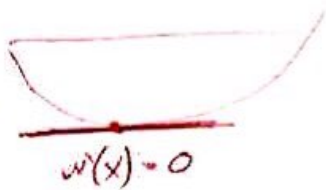


Find deflection at $x = 3$

① $12 - 9 = 3$
 $w(x) =$

②

$w(x) = 4x^3 + 2x^2 + 5x$ find ~~at~~
 at which max deflection

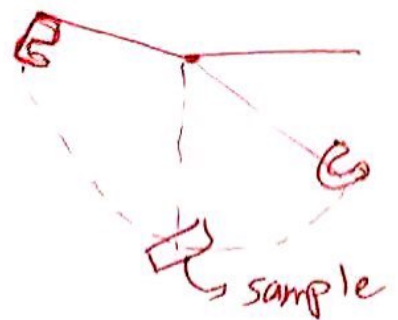


$$w'(x) = 12x^2 + 4x + 5 = 0$$

$$x =$$

4

Pendol is released from rest
final height 1.6m if you
know that friction 3 J
and Max velocity 5 m/s
find toughness



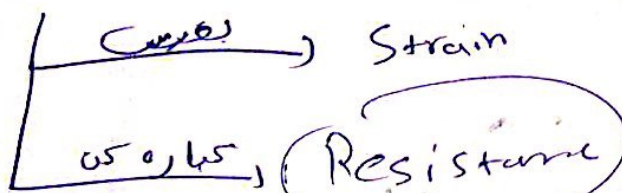
→ h_1, h_2 ∴ $U = mg(h_1 - h_2) - \text{friction}$

energy
conservation

$$h_1 = \frac{v^2}{2g}$$

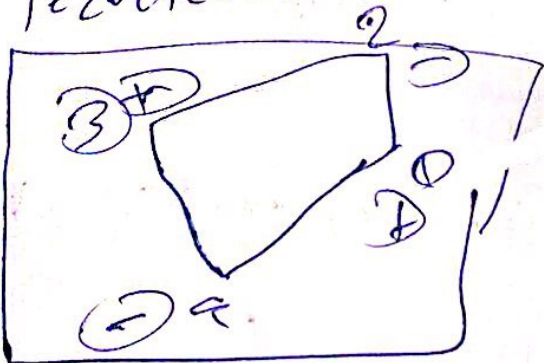
Strain gauge

8



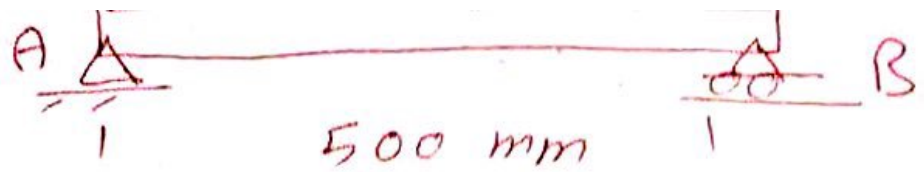
Resistance 11 نوع

Piezoelectric



Wheatstone Bridge

Voltage



a) Reaction $R_A = 120 \text{ N}$
 $R_B = 80 \text{ N}$

b) Max Moment
 $M = 24$

c) Max bending stress

$$\sigma = \frac{M \cdot c}{I} = \frac{M (5 \times 10^{-3})}{\frac{1}{12} b h^3 \times 12 \times 12} =$$

